

ADJUSTABLE LOCKNUTS

SPIETH



- **POSITIVE LOCKING WITHOUT WASHERS AND KEYS**
- **SUPPORTS EXTREME THRUST LOADS**

General:

Spieth Adjustable Locknuts adjust, preload, and secure bearings and other machine components on shafts and spindles with a high degree of stiffness. They permit the correct setting of the axial preload of bearings and ensure that the working loads are carried safely. The locknuts are secured positively without additional components, keys, or extra machining. The high load-carrying ability of the locknut is achieved through full 360° contact around the thread flanks guaranteeing a vice-grip locking effect. This contact around the full circumference ensures that the locknut will not loosen, even when subjected to extreme shock loads. This is an important feature, since locking with many commercially available locknuts relies on only one or a few "pinch" points on the circumference of the thread. Another advantage of the SPIETH locknut is the feature of aligning its contact face either with zero runout or, if necessary, adjusted to suit inaccuracies of in-line connecting components, thus compensating for minor errors of parallelism due to the stack-up of mating parts. Spindle

runout, due to uneven bearing loading, can therefore be eliminated. A high degree of accuracy can be maintained even when repetitive tightening and loosening are required. The benefits of the SPIETH locknuts offer solutions for a wide range of general precision, heavy machinery, and equipment applications. The unique locking capability, with high stiffness and accuracy, is excellent for spindle applications - especially suited for high speed and/or high dynamically-loaded spindles or mechanisms.

Thread sizes are equivalent to the standard bearing locknut series N/AN American Standard National Class 3 Fine, and metric sizes to ISO 5H fit. The widths approximate two N/AN locknuts. SPIETH SERIES MSW heavy-duty ADJUSTABLE LOCKNUTS are applied in conjunction with combined needle and roller thrust bearings, plain thrust bearings, or taper roller bearings, on ballscrews, worm drives, bevel gears, and other high-thrust-carrying components.

Design:

Two axially-arranged radial grooves, one from the outside and one from the inside, result in a diaphragm-like cross-section which gives the nut a predetermined axial elasticity. The internal groove divides the nut into a clamping and locking section. Depending on the nut size, three to eight axially arranged socket head cap screws are used to clamp the sections together. This eliminates the thread flank clearance between the shaft thread and locknut and allows for shockproof locking. Depending on the size and load capacity, two types of locknuts are available. For all MSR, MSA, and MSW sizes up to locknut 70.46, the axial bearing preload is adjusted by tightening the locknut through direct face contact. Size MSW 70.60 and larger use a number of clamping screws to reduce friction on the nut contact face, thus allowing for accurate preload adjustment. MSR Locknuts are made of alloy steel, MSW Locknuts are made of heat-treated alloy steel. Both locknut series have a black oxide finish for rust protection. The nut thread is manufactured to American Standard Class 3 Fine or ISO H5 fit. The contact face is machined square to the threads within .0002 in. Both types of locknuts are symmetrical in construction and are manufactured from

solid stock. Threads and contact faces are machined by special single-point threading tools in one set-up to guarantee squareness. Balancing, therefore, is required only in extreme high-speed cases. The SPIETH Locknut's main feature is its capability to lock onto the threads over the entire 360-degree circumference.

PRELOADING NUTS: All MSR, MSA sizes, and MSW sizes up to 70.46:

These nuts have holes or slots on the outer diameter, which are suitable for available radial spanner wrenches. When the nuts are used in recessed applications, a tubular face spanner wrench with exposed prongs for MSW series and pins for MSR or MSA series is recommended. See Fig. 21 on page 10.

PRELOADING NUTS: Size MSW 72.60 and larger:

These nuts are screwed into position either by hand or by an available hook spanner wrench and locked into position without making face contact. Hardened clamping screws are then used axially for preloading; which, after adjusting, are secured in position by radial set screws through aluminum slugs. See Fig. 2 on page 3.

Mating Component:

MATING THREAD:

The shaft thread must be manufactured to American Class 3 "fine" or ISO Class 4h if applied for high-precision spindles or class "medium" or ISO class 6g for general applications. Surface roughness must be low to avoid compaction during shock loads.

MATING PARTS FOR LOCKNUTS UP TO SIZE MSW 70.46:

The preload torque is applied directly to the nut. However, means must be provided on the shaft to counter the torque. For optimum results, the accuracy of components assembled between the shaft, shoulder, and locknut contact face (bearing, gears, spacers, etc.) must be parallel and square to the bore within .0005" or better. In addition, the surface finish of the assembled

components should be a minimum of 32 micro-inches or better in order to avoid surface compaction under load.

MATING PARTS FOR NUTS SIZE MSW 72.60 AND LARGER:

The axial preload applied through the hardened clamping screws must not act directly on the bearing face or soft surfaces. A thrust ring at least as thick as the bearing race should be sandwiched in between the bearing and locknut. The ring must be hardened, parallel, and square as specified above. If the design requires other components such as gears to be placed between the bearing and the locknut, again the accuracy must be held as specified above to avoid excessive deflection and surface compaction. If these components are hardened, the clamping screws can press directly against them without a thrust ring.

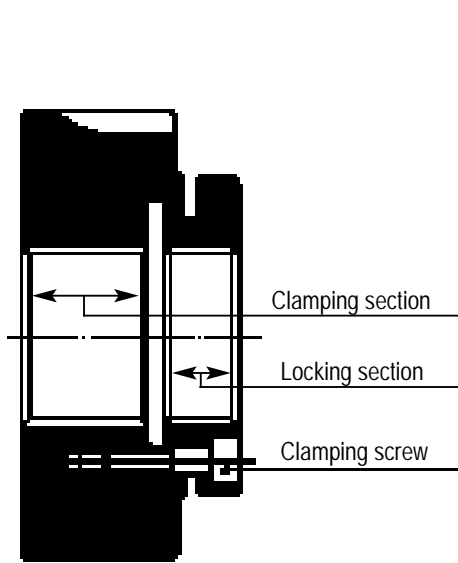


Fig. 1

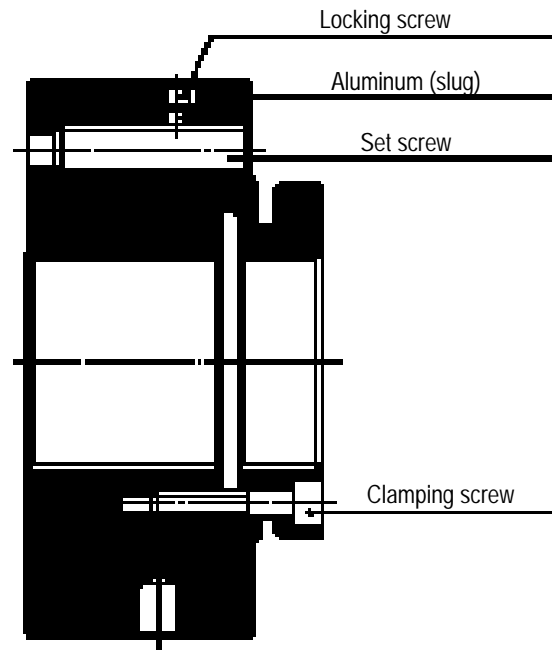


Fig. 2

Setting an axial pretensioning force:

The axial pretension of a threaded connection often plays a decisive role in successful function and must therefore be set with accuracy. However, in most assembly shops, direct measurement of this variable is not possible, making indirect methods of setting necessary. For this purpose, the locknut tightening torque corresponding to the required pretensioning force is calculated. This factor can be determined using the following equation.

For all MSR, MSA and MSW sizes up to 70.46:

$$M_v = (F_v + B) (A + \mu_A \cdot r_A) 10^{-3}$$

M_v = Tightening torque of the locknut [Nm]

F_v = Required axial pretensioning force of the screw connection [N]

B = Locknut-specific allowance [N], compensates face end relief due to the locking process

A = Constant [mm], includes the calculation factors for the respective thread diameters (see table)

μ_A = Coefficient of friction for the face contact surface of the locknut.
Approximate value $\mu_A = 0.1$ steel/steel

r_A = Effective mean friction radius for the face contact surface of the locknut [mm]

The locking process places the shaft thread under stress and develops intensive surface contact (=high axial rigidity). At the same time, this serves to relieve tension on the face contact surface of the locknut. This effect can be simply compensated for by increasing the tightening torque at assembly. This higher tightening torque is determined by using the allowance B relative to the required pretensioning force F_v .

For MSW size 72.60 and larger:

The tightening torque for the set screw is determined according to the following formula:

$$M_D = \frac{F_v}{8} (A + \mu_D \cdot \frac{d_6}{4}) 10^{-3}$$

M_D = Tightening torque per set screw [Nm]

F_v = Required axial pretensioning force of the screw connection [N]

A = Constant [mm], includes the calculation factors for the respective thread diameters (see table)

μ_D = Coefficient of friction for the face contact surface of the set screw.
Approximate value = 0.13

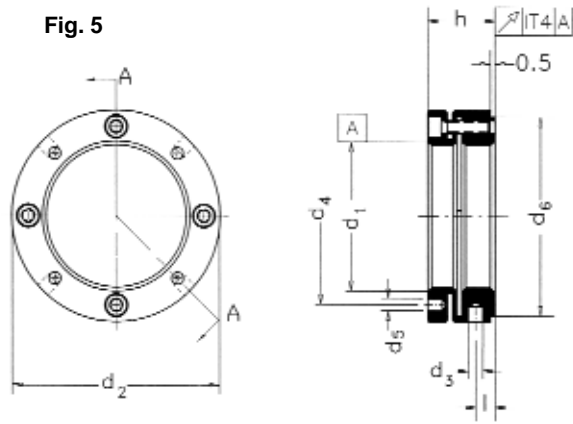
d_6 = Dog point dia. of the set screw [mm] (see table)

Fig. 6

1 N	≅	0.225 lbs	
1 Nm	≅	8.8583 in-lbs	
1 inch	≅	25.4 mm	
kg cm ²	=	8.8583 · 10 ⁻⁴	in-lbs-sec ²

SPIETH Locknuts Series MSA - Metric

Fig. 5



Designation of a locknut with
 $d_1 = M 20 \times 1$:
 Locknut MSA 20 x 1

The allowable axial loads specified in the table are guideline values calculated with a safety margin of 1.6

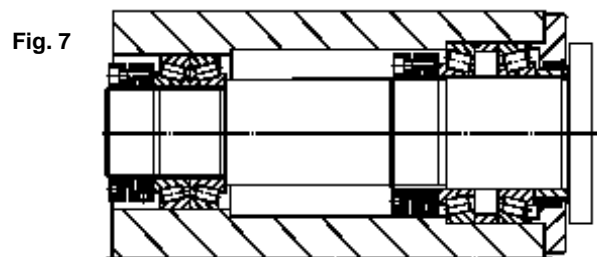
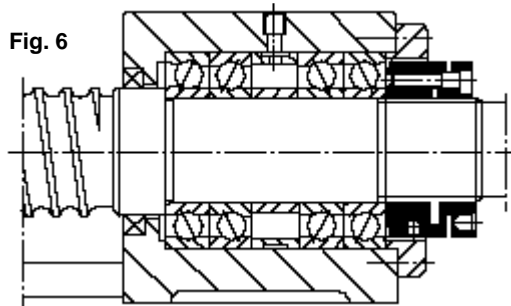
- under static load, based on the minimum yield point,
- under dynamic loads, based on the minimum fatigue strength.

Series MSA Locknuts with reduced contact surface and in some cases smaller outside diameters relative to the MSR series are particularly suited for mounting angular ball bearings and cylinder roller bearings of ISO diameter series 9.

MSA size ($d_1 \times$ pitch)	Dimensions in mm							Clamp screws			Calculation factor A	Locknut-specific allowance B	Allowable axial load		Mass moment of inertia J kg cm ²
	d_2	$d_3^{(1)}$	d_4	$d_5^{(1)}$	d_6	h	l	Sizes	n	T_s			dyn.	stat.	
								SHCS	Qty	Nm	mm	kN	kN		
20x1	35	4	27.5	3.2	31	17	5	M 3x12	5	2	1.281	3938	23	31	0.142
25x1.5	40	4	32.5	3.2	36	19	6.5	M 3x14	5	2	1.633	3859	35	49	0.265
30x1.5	45	5	37.5	3.2	41	19	6.5	M 3x14	5	2	1.921	3780	39	56	0.400
35x1.5	53	5	45.5	4.3	48	22	7	M 4x16	4	2.9	2.210	3666	47	66	0.904
40x1.5	58	5	50.5	4.3	54	22	7	M 4x16	4	2.9	2.500	3588	50	68	1.242
45x1.5	64	6	54	4.3	59	23	7	M 4x16	5	2.9	2.789	4388	58	78	1.888
50x1.5	69	6	59	4.3	64	24	8	M 4x16	6	2.9	3.079	5148	63	85	2.563
55x1.5	73	6	64	4.3	69	24	8	M 4x16	6	2.9	3.369	5031	59	79	3.001
60x1.5	78	6	69	4.3	74	24	8	M 4x16	6	2.9	3.655	4914	61	81	3.758
65x1.5	83	6	74	4.3	79	24	8	M 4x16	7	2.9	3.948	5597	94	124	4.611
70x1.5	93	8	83	5.3	88	27	9	M 5x20	6	6	4.238	7620	136	178	9.094
75x1.5	98	8	88	5.3	93	27	9	M 5x20	6	6	4.525	7430	138	183	10.866
80x2	103	8	93	5.3	98	28	10	M 5x20	6	6	4.873	7239	148	196	13.397
85x2	112	8	100	6.4	106	30	10	M 6x20	6	10	5.168	9990	172	228	21.260
90x2	117	8	105	6.4	111	30	10	M 6x20	6	10	5.453	9720	174	230	24.650
95x2	122	8	110	6.4	116	30	10	M 6x20	6	10	5.744	9450	176	232	28.384
100x2	130	8	118	6.4	123	32	11	M 6x22	6	10	6.033	9180	205	271	38.620
105x2	135	8	123	6.4	128	32	11	M 6x22	6	10	6.321	8910	207	274	43.852
110x2	140	8	128	6.4	133	32	11	M 6x22	6	10	6.616	8640	212	280	49.539
120x2	155	8	140	6.4	145	36	13	M 6x25	6	10	7.193	8100	308	408	89.148
130x3	165	8	153	6.4	155	36	13	M 6x25	6	10	7.895	7560	306	405	109.890
140x3	180	10	165	6.4	170	36	12	M 6x25	8	10	8.475	9360	359	476	160.150
150x3	190	10	175	6.4	180	36	12	M 6x25	8	10	9.050	8640	369	489	191.977
160x3	205	10	185	8.4	195	40	14	M 8x30	8	25	9.633	14520	417	552	300.080
170x3	215	10	195	8.4	205	40	14	M 8x30	8	25	10.213	13200	423	560	351.919
180x3	230	10	210	8.4	220	40	14	M 8x30	8	25	10.789	11880	489	648	475.748
190x3	240	10	224	8.4	230	40	14	M 8x30	8	25	11.362	10560	495	656	548.328
200x3	245	10	229	8.4	235	40	14	M 8x30	8	25	11.948	9240	436	578	542.596

1) In same quantity as clamp screws in Fig. 6 and Fig. 7.
 Spanner wrenches available upon request.

Specifications and dimensions are subject to change without notice.
 Specials available upon request.



This locknut provides a high degree of axial rigidity for the bearing assembly of a ballscrew. The locknut offers a particular advantage for fluctuating dynamic conditions.

Tapered roller bearings require minimum runout and provide high axial rigidity. A radial force applied to tapered roller bearings will generate an axial force on the connecting components. Even if the preload is lost, the locknut remains safely locked.

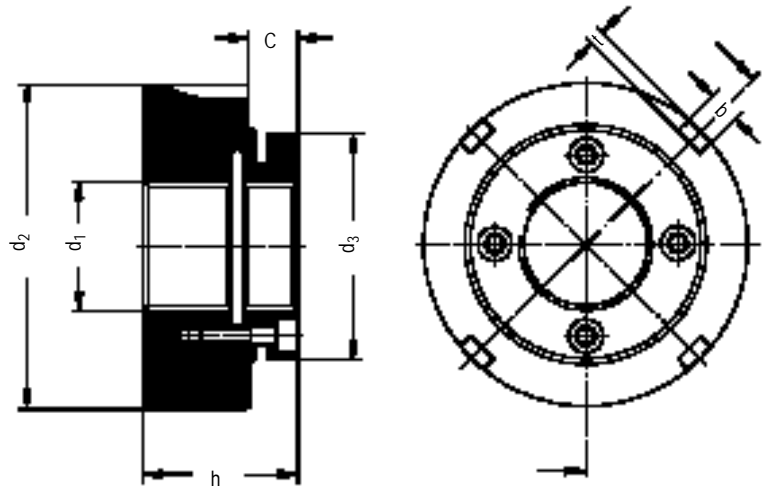
SPIETH Locknuts Series MSW - Metric

Fig. 8

Designation of a locknut with
 $d_1 = M 30 \times 1.5$ and length = 28 mm:
 Locknut MSW 30.28

The allowable axial loads specified in the table are guideline values calculated with a safety margin of 1.6

- under static load, based on the minimum yield point,
- under dynamic loads, based on the minimum fatigue strength.

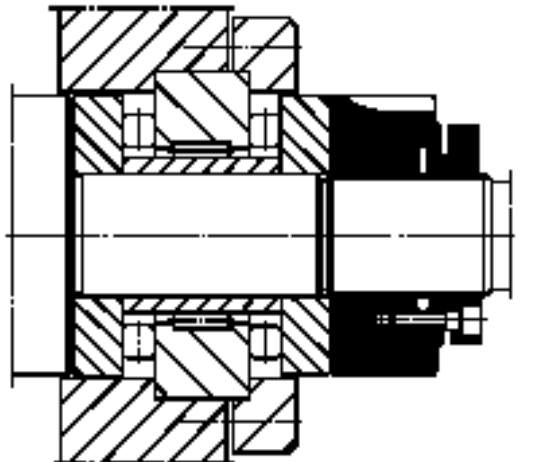


MSW size (thread x length)	d_1 ($d_1 \times$ pitch) ISO - 5H	Dimensions in mm							Clamp screws			Calculation factor A mm	Locknut-specific allowance B N	Allowable axial load		Mass moment of inertia J kg cm ²
		d_2	d_3	h	C	$b^{1)}$	t	Sizes	n	T_s						
								SHSC	Qty	Nm						
		dyn.	stat.													
LIGHT SERIES	20 x 28	M 20x1.5	42	38	28	11	6	2.5	M 4x16	4	2.9	1.344	1560	57	80	0.4860
	25 x 28	M 25x1.5	47	43	28	11	7	3	M 4x16	4	2.9	1.633	1560	68	102	0.7419
	30 x 28	M 30x1.5	52	48	28	11	7	3	M 4x16	4	2.9	1.921	1560	77	123	1.0853
	35 x 28	M 35x1.5	60	53	28	11	8	3.5	M 4x16	4	2.9	2.210	1560	88	144	1.8034
	40 x 28	M 40x1.5	65	58	28	11	8	3.5	M 4x16	4	2.9	2.500	1560	97	165	2.4259
	45 x 28	M 45x1.5	70	63	28	11	8	3.5	M 4x16	6	2.9	2.789	2340	105	184	3.1432
HEAVY SERIES	50 x 32	M 50x1.5	75	68	32	11	8	3.5	M 4x16	6	2.9	3.079	2340	147	267	4.7785
	20 x 40	M 20x1.5	52	42	40	11	7	3	M 4x20	4	2.9	1.344	936	110	156	1.7401
	25 x 40	M 25x1.5	62	47	40	11	8	3.5	M 4x20	4	2.9	1.633	936	131	196	3.4125
	30 x 44	M 30x1.5	68	57	44	11	8	3.5	M 4x20	4	2.9	1.921	936	172	273	5.5377
	35 x 44	M 35x1.5	73	60	44	11	8	3.5	M 4x20	4	2.9	2.210	936	195	320	7.4069
	40 x 44	M 40x1.5	75	62	44	11	8	3.5	M 4x20	4	2.9	2.500	936	215	367	7.9830
	45 x 44	M 45x1.5	90	70	44	11	10	4	M 4x20	6	2.9	2.789	1404	234	410	16.4246
	50 x 46	M 50x1.5	95	75	46	11	10	4	M 4x20	6	2.9	3.079	1404	268	488	21.3395
	55 x 46	M 55x1.5	100	80	46	12	10	4	M 5x25	6	6.0	3.369	2286	272	504	23.5948
	60 x 46	M 60x1.5	100	85	46	12	10	4	M 5x25	6	6.0	3.655	2286	294	551	24.7692
	65 x 46	M 65x1.5	110	90	46	12	10	4	M 5x25	6	6.0	3.948	2286	314	598	35.8605
	70 x 46	M 70x1.5	115	95	46	12	10	4	M 5x25	6	6.0	4.238	2286	333	645	42.2151

1) In same quantity as clamp screws.
 Spanner wrenches available upon request.

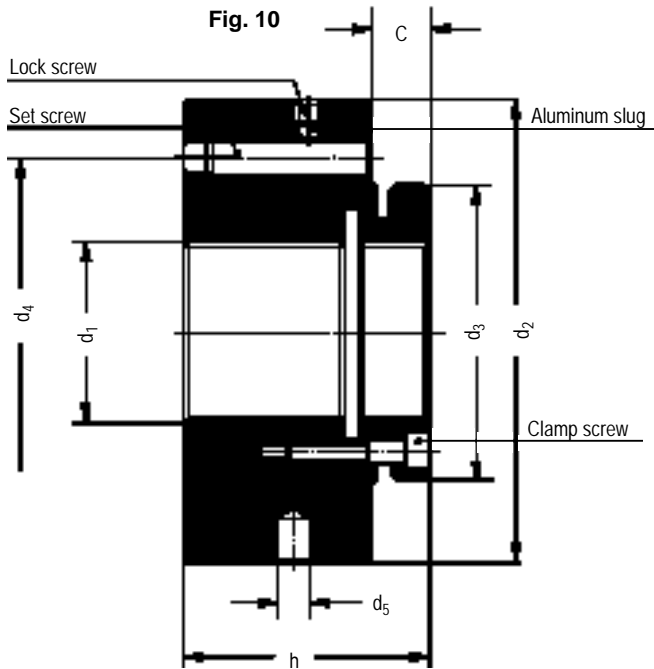
Specifications are subject to change without notice.
 Specials available upon request.

Fig. 9



The high loads are reliably transmitted to the feed drive through the rigid locknut. The excellent locking properties of the locknut are of major importance under dynamic loads.

SPIETH Locknuts Series MSW - Metric, Large



Designation of a locknut with
 $d_1 = M 72 \times 1.5$ and length = 60 mm:
 Locknut MSW 72.60

The allowable axial loads specified in the table are guideline values calculated with a safety margin of 1.6

- under static load, based on the minimum yield point,
- under dynamic loads, based on the minimum fatigue strength.

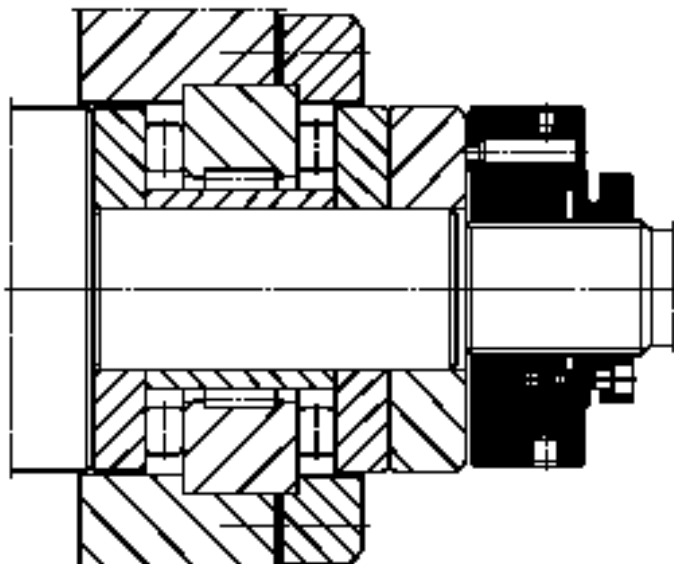
MSW size (thread x length)	Dimensions in mm						d_5	Clamp screws			Allowable axial load		Mass moment of inertia J	
	d_1 ($d_1 \times$ pitch)	d_2	d_3	h	C	d_4		Size	n_1	T_s	dyn.	stat.		
	ISO - H5							Qty	SHCS	Qty	Nm	kN	kN	kg cm ²
72 x 60	M 72x1.5	135	95	60	14	105	8	4	M 5x25	6	6	468	749	110.481
85 x 60	M 85x2	160	110	60	14	124	8	4	M 6x30	6	10	807	1050	218.131

MSW size (thread x length)	Preload set screw					Lock screw		Aluminum slug		
	Size	n	d_6	M_D ¹⁾	Calculation factor A	Size	n_2	\emptyset	Length	n_3
	SHCS	Qty	mm	Nm	mm	SHCS	Qty	mm	mm	Qty
72 x 60	M 10x45	8	7	34	0.76575	M 6x8	8	4.5	3	8
85 x 60	M 12x45	8	8.5	60	0.91282	M 8x8	8	6	3	8

1) Recommended tightening torque by the screw manufacturer.
 Spanner wrenches available upon request.

Subject to changes.
 Specials available upon request.

Fig. 11



Typical Application

Because of friction, the larger MSW locknuts cannot produce the required axial preload using the nuts' own tightening torque. Axial preload is provided with the set-screws in the nut body.

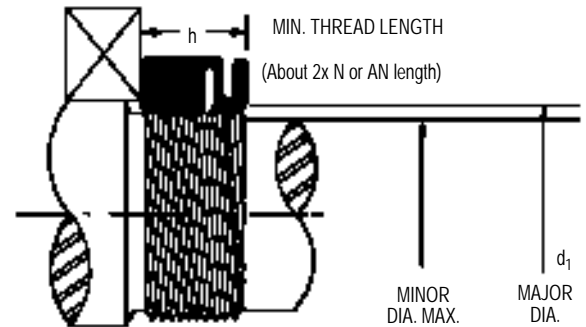
MALE THREAD DATA

AMERICAN NATIONAL STD. FINE-CLASS 3 FIT
Limiting Dimensions for External Threads (Bolts)
to Suit Spieth Adjustable Locknuts

Dimensions in Inches

SPIETH MSR Locknut	Equiv. lock-nut no.	Major dia.		Minor dia.-max.	Pitch dia.		h
		Max.	Min.		Max.	Min.	
Special	N-00	.391	.386	.3527	.3707	.3681	.75
Special	N-01	.469	.464	.4307	.4487	.4461	.75
Special	N-02	.586	.581	.5477	.5657	.5627	.75
17x32	N-03	.664	.659	.6257	.6437	.6407	.75
20x32	N-04	.781	.776	.7427	.7607	.7573	.75
25x32	N-05	.969	.964	.9307	.9487	.9453	.75
30x18	N-06	1.173	1.165	1.1048	1.1369	1.1329	.81
35x18	N-07	1.376	1.368	1.3078	1.3399	1.3359	.88
40x18	N-08	1.563	1.555	1.4948	1.5269	1.5224	.88
45x18	N-09	1.767	1.759	1.6988	1.7309	1.7264	.88
50x18	N-10	1.967	1.959	1.8988	1.9309	1.9264	1.06
55x18	N-11	2.157	2.149	2.0888	2.1209	2.1158	1.06
60x18	N-12	2.360	2.352	2.2918	2.3239	2.3188	1.06
65x18	N-13	2.548	2.540	2.4798	2.5119	2.5068	1.12
70x18	N-14	2.751	2.743	2.6828	2.7149	2.7098	1.12
75x12	AN-15	2.933	2.922	2.8308	2.8789	2.8735	1.12
80x12	AN-16	3.137	3.126	3.0348	3.0829	3.0770	1.28
85x12	AN-17	3.340	3.329	3.2378	3.2859	3.2785	1.28
90x12	AN-18	3.527	3.516	3.4248	3.4729	3.4655	1.28
95x12	AN-19	3.730	3.719	3.6278	3.6759	3.6685	1.28
100x12	AN-20	3.918	3.907	3.8158	3.8639	3.8565	1.28
105x12	AN-21	4.122	4.111	4.0198	4.0679	4.0596	1.28
110x12	AN-22	4.325	4.314	4.2228	4.2709	4.2626	1.28
120x12	AN-24	4.716	4.705	4.6138	4.6619	4.6536	1.50
130x12	AN-26	5.106	5.095	5.0038	5.0519	5.0436	1.50
140x12	AN-28	5.497	5.486	5.3948	5.4429	5.4346	1.50
150x12	AN-30	5.888	5.877	5.7858	5.8339	5.8256	1.50
160x 8	AN-32	6.284	6.269	6.1306	6.2028	6.1937	1.62
170 x 8	AN-34	6.659	6.644	6.5056	6.5778	6.5687	1.62
180x 8	AN-36	7.066	7.051	6.9126	6.9848	6.9757	1.62
190x 8	AN-38	7.472	7.457	7.3186	7.3908	7.3817	1.62
200x 8	AN-40	7.847	7.832	7.6936	7.7658	7.7544	1.62

Fig. 13



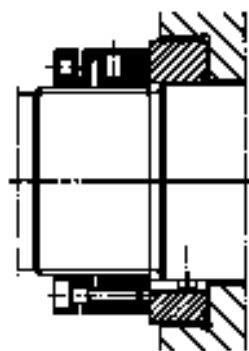
SPIETH MSR Locknut	Major dia.		Minor dia.-max.	Pitch dia.		h
	Max.	Min.		Max.	Min.	
5/8"-18	.625	.6168	.5568	.5889	.5859	.75
3/4"-16	.750	.7410	.6733	.7094	.7062	.75
7/8"-14	.875	.8652	.7874	.8286	.8250	.75
1"-12	1.000	.9886	.8978	.9459	.9415	.75
1-1/8"-12	1.125	1.1136	1.0228	1.0709	1.0664	.81
1-1/4"-12	1.250	1.2388	1.1478	1.1959	1.1919	.88
1-3/8"-12	1.375	1.3638	1.2728	1.3209	1.3169	.88
1-1/2"-12	1.500	1.4888	1.3978	1.4459	1.4419	.88
1-5/8"-12	1.625	1.6138	1.5228	1.5709	1.5664	.88
1-3/4"-12	1.750	1.7388	1.6478	1.6959	1.6913	.88
1-7/8"-12	1.875	1.8638	1.7728	1.8209	1.8163	.88
2"-12	2.000	1.9888	1.8978	1.9459	1.9412	1.06
2-1/4"-12	2.250	2.2388	2.1478	2.1959	2.1911	1.06
2-1/2"-12	2.500	2.4888	2.3978	2.4459	2.4410	1.12
2-3/4"-12	2.750	2.7388	2.6478	2.6959	2.6909	1.12
3"-12	3.000	2.9888	2.8978	2.9459	2.9408	1.12

Fig. 14



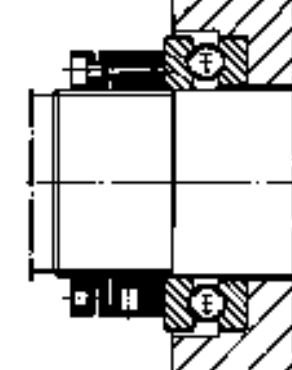
Axial positioning of spring on shaft.

Fig. 15



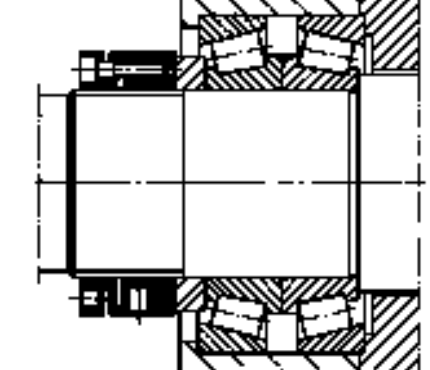
Axial play adjustment on thrust washer.

Fig. 16



Adjustment of axial thrust bearings.

Fig. 17



Ball screw end mounting.

ASSEMBLY OF SPIETH ADJUSTABLE LOCKNUTS:

Note that the locking screws can only be tightened when the nut is engaged entirely on the male thread to avoid distortion of the locknut. Clean and oil the thread before engaging the nut. Once a Spieth locknut has been preloaded and locked on a thread, it should not be put on another thread. Otherwise, high adjustment accuracy might not be achievable.

For all MSR, MSA and up to size 70.46 MSW Locknuts: With locking screws D loose, screw the locknut on to the shaft thread C until it is within .100" of the mating face E (Fig. 18).

With an Allen wrench, slightly tighten each locking screw D crosswise and uniformly until the locknut is barely turnable by hand (the flank clearance is nearly eliminated). In this condition, the centerline of the locknut has coincided with the centerline of the spindle, the mating face A is perpendicular to the common centerline, and the thread flanks of the locknut are in uniform contact over the full circumference of both the clamping and locking section (Fig. 19).

With a spanner wrench, tighten the locknut against the mating face E, with three to five times the required

preload to prestretch the threads, eliminating any subsequent compaction of nut, shaft threads and contact surfaces. (Radial or face spanner wrenches are available upon request). Then loosen the locknut and torque to the desired preload.

Secure the locknut in position by tightening the locking screws D gradually, evenly, and crosswise. Face and radial runout of the mating component caused by summation of small errors in parallelism in the connecting parts can be eliminated by individually adjusting the locking screws (Fig. 23).

To release a SPIETH Adjustable Locknut, slightly loosen the locking screws D crosswise before completely releasing all screws to avoid binding the last screw.

The locknut can be easily tightened or loosened by means of a modified Allen wrench or by a special tube wrench with prongs or pins when assembly and disassembly must take place axially. (Fig. 21, 22)

For heavy preload, use radial spanner holes rather than face spanner holes to avoid deflection of the membrane of the locknut.

Fig. 18

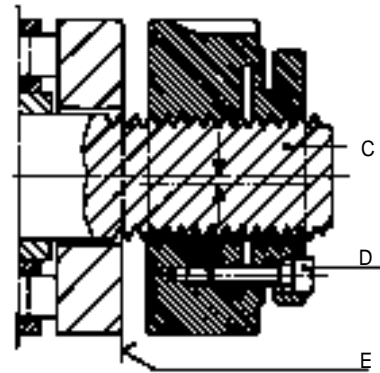


Fig. 19

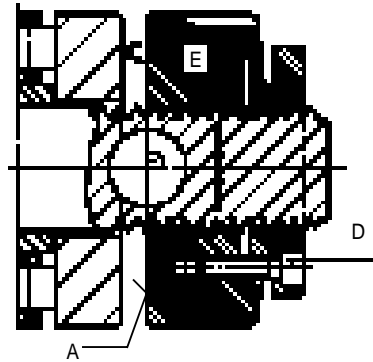


Fig. 20

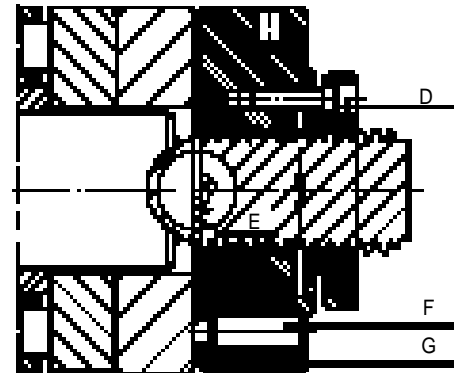
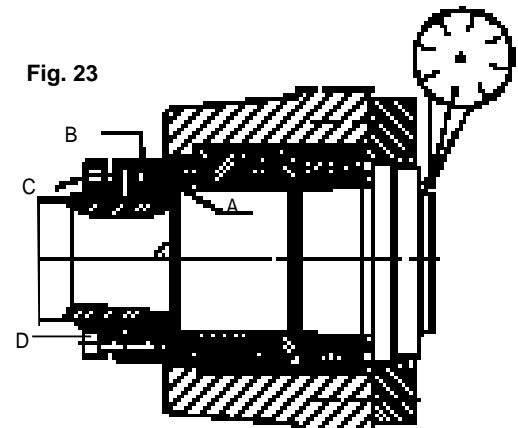


Fig. 23



ASSEMBLY INSTRUCTIONS FOR SIZE MSW 72.60 AND LARGER: (Large Series)

Follow instructions in paragraphs 2 and 3 above.

At this point, the locknut is at a distance of about .1" from the mating face E. Secure by tightening the locking screws D to the required T_S torque. Then tighten the clamping screws F against the hardened mating face E crosswise - - initially three to five times the required torque. This eliminates any subsequent compaction of nut, spindle threads and contact surfaces. Loosen the clamping screws F again and subsequently tighten to the required MD preload

torque. Finally, tighten the set screws G and check the locking screws D once more to ensure the required T_S torque setting. (Fig. 20)

To release the locknut, first loosen the set screws G, then slightly loosen the clamping screws F in sequence and only subsequently loosen these screws completely in a crosswise manner. This procedure avoids binding of the last screw. Release clamp screws D in a crosswise manner.

Fig. 21

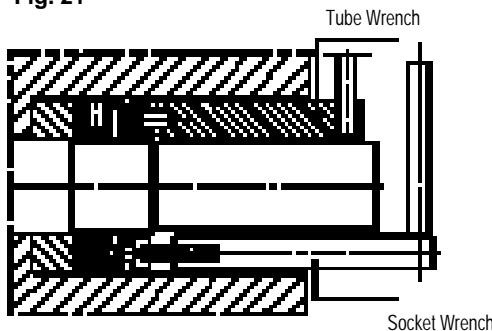
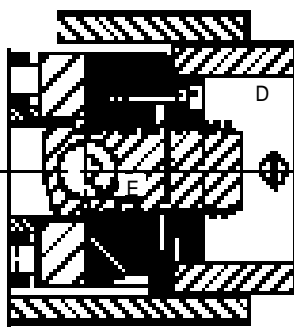


Fig. 22

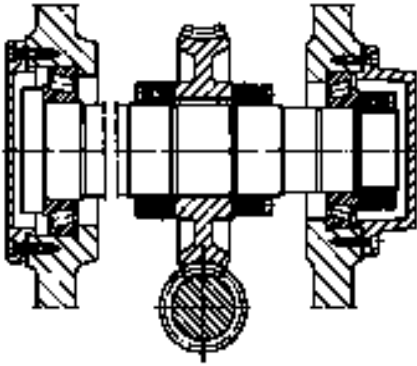


Special wrenches for tightening clamp screws on recessed locknuts.

Adjustment of locknut to compensate for minor inaccuracy of mating parts.

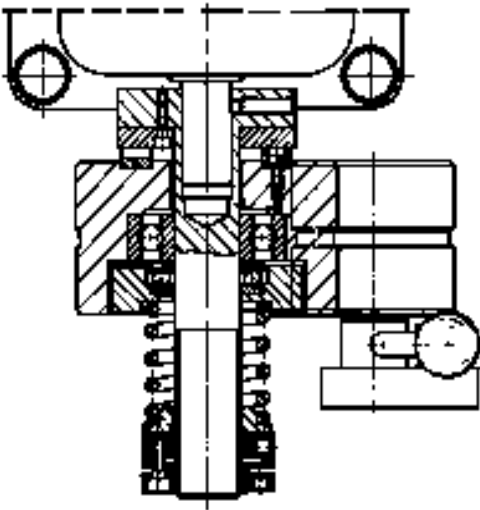
APPLICATIONS FOR SPIETH ADJUSTABLE LOCKNUTS

Fig. 24



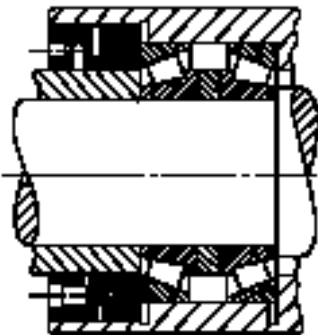
Adjustment of worm gear and bearings.

Fig. 27



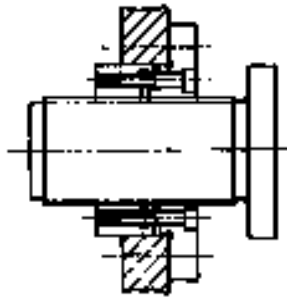
Fine adjustment and reliable locking are essential requirements on this slipclutch assembly of a fully automated zipper assembly machine. The SPIETH Locknut, incorporating such features, permits positioning and locking in any axial location on the threaded shaft.

Fig. 29



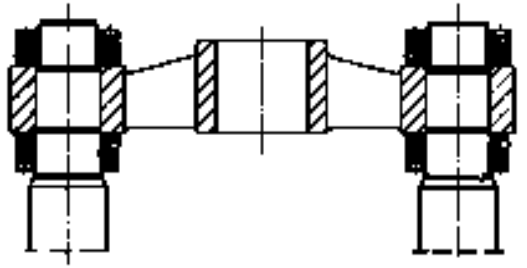
Special SPIETH Adjustable Lockscrew for preloading taper ball bearings.

Fig. 25



SPIETH special flange-type locknut with adjustable positioning of stop screw. Locknut is locked positively in any position.

Fig. 26

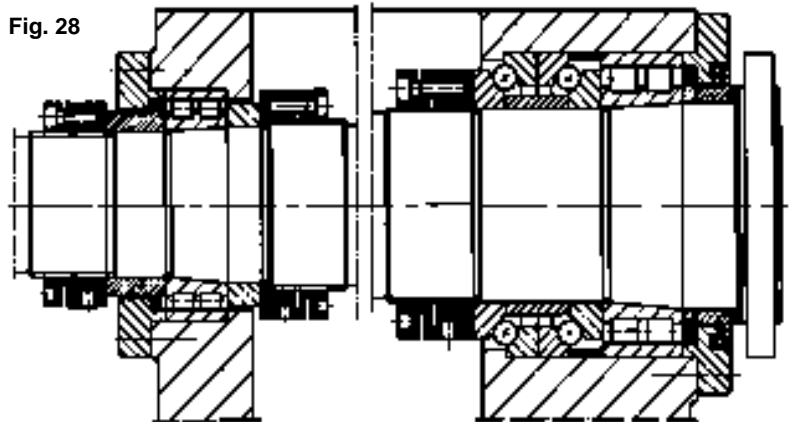


SPIETH Locknuts applied to provide adjustable support surfaces for a four-post ram plate.

The distance and the parallelism are accurately established by means of backlash-free locknuts.

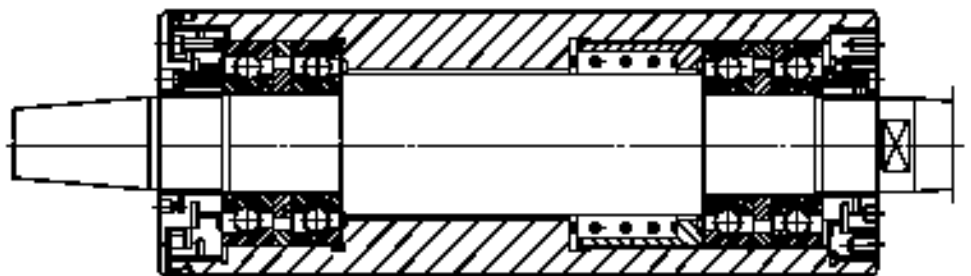
Squareness of the nut contact faces are easily adjusted. Deflection and misalignment of the guide posts will therefore be eliminated when the counter nuts are tightened.

Fig. 28




Head spindle assembly with double row roller and thrust bearings: Adjustment of axial preload accomplished with SPIETH Adjustable Locknuts.

Fig. 30



Special SPIETH Lockscrews incorporating such features as labyrinth grooves and other configurations may be supplied to satisfy specific machine designs.


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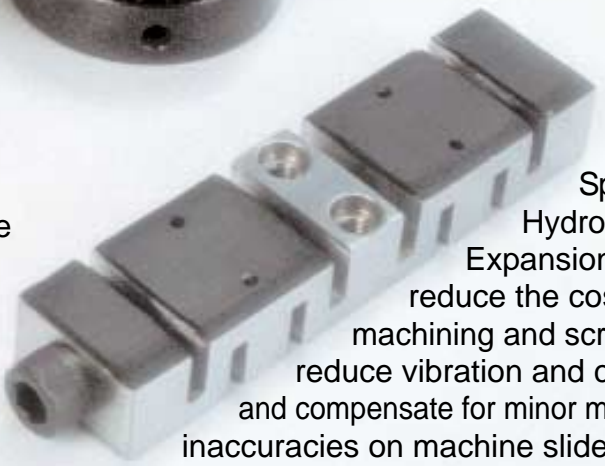


Spieth precision Adjustable Locknuts provide vise-like locking without washers and keys.



Spieth Adjustable Hydrodynamic Guide Bushings provide accurate guiding, adjustable guide clearance, hydrodynamic lubrication effect. Linear and rotary applications.

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